
Supplementary information: full mathematical details to
The Graetz-Nusselt problem extended to continuum flows with finite slip

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1 GOVERNING EQUATION

The Graetz-Nusselt problem considers transport of heat between a fluid and a tube, beginning at some location in the hydrodynamically developed flow region. The tube entrance length is unheated, and the wall temperature T_1 of the heated or cooled section is constant but different from that of the entering fluid temperature T_0 . The fluid has constant physical properties, and viscous dissipation and axial heat conduction are neglected. In this study we extend this classical problem to fluid flows with a finite slip velocity u_s at the wall at $r = R$. Wall slip can be quantified by a slip length b , which is defined according to Navier's slip condition,

$$u_s = -b \left. \frac{\partial u}{\partial r} \right|_{r=R}. \quad (1.1)$$

The governing equation describing stationary heat transport in such an axisymmetric cylindrical system, under the assumption of constant density ρ and thermal conductivity k , can be written as

$$u \frac{\partial T}{\partial x} = \frac{\alpha}{r} \frac{\partial}{\partial r} \left(r \frac{\partial T}{\partial r} \right), \quad (1.2)$$

where $u(r, b)$ describes the velocity profile of the laminar fluid flowing in the x -direction, and $\alpha = k/(\rho C_p)$ is the thermal diffusivity. The initial and boundary condition are $T(0, r) = T_0$ and $T(x, R) = T_1$.

Solving the Navier-Stokes equation for stationary slip flow in the axial direction, again assuming constant ρ and k , yields the following expression for the velocity profile,

$$\tilde{u} = \frac{2(1 - \tilde{r}^2) + 4\tilde{b}}{1 + 4\tilde{b}}, \quad (1.3)$$

where $\tilde{u} = u/u_{av}$, $\tilde{r} = r/R$, and $\tilde{b} = b/R$. \tilde{u} can be written as the sum of the variable velocity $\tilde{u}_v(\tilde{r}, \tilde{b})$ and the slip velocity $\tilde{u}_s(\tilde{b})$ at the wall:

$$\tilde{u} = \tilde{u}_v + \tilde{u}_s = \frac{2(1 - \tilde{r}^2)}{1 + 4\tilde{b}} + \frac{4\tilde{b}}{1 + 4\tilde{b}}. \quad (1.4)$$

The governing heat equation can now be non-dimensionalised using $\Theta = (T_1 - T)/(T_1 - T_0)$ and $\tilde{x} = x/L$ (L being the length of the heated or cooled section of the pipe),

$$\frac{(1 - \tilde{r}^2) + 2\tilde{b}}{2(1 + 4\tilde{b})} \frac{\partial \Theta}{\partial (\tilde{x}/Gz)} = \frac{1}{\tilde{r}} \frac{\partial}{\partial \tilde{r}} \left(\tilde{r} \frac{\partial \Theta}{\partial \tilde{r}} \right), \quad (1.5)$$

with $\Theta(0, \tilde{r}) = 1$ and $\Theta(\tilde{x}, 1) = 0$. The Graetz number Gz is defined as

$$Gz = RePr \frac{D}{L} = 4 \frac{u_{av} R^2}{\alpha L}, \quad (1.6)$$

where $Re = u_{av} D/\nu$ is the Reynolds number, $Pr = \nu/\alpha$ is the Prandtl number, and D is the diameter of the tube. For $\tilde{x}/Gz < 0.01$ the fluid flow is thermally developing, while for $\tilde{x}/Gz > 0.1$ the fluid flow is said to be thermally developed.

2 NUSSELT NUMBER

The local Nusselt number Nu_x is defined, by definition, as

$$Nu_x = \frac{h_x D}{k}, \quad (2.1)$$

where h_x is the local heat transfer coefficient, being a function of the dimensionless position \tilde{x}/Gz . Following Newton's law of cooling and using Fourier's Law of thermal conduction, the heat transfer coefficient h_x can be rewritten as

$$h_x = \frac{q''}{\Delta T} = -\frac{k}{\langle T \rangle - T_1} \left. \frac{\partial T}{\partial r} \right|_{r=R}, \quad (2.2)$$

where q'' represents the local heat flux between the wall and the fluid. Rewriting the temperature gradient in dimensionless form gives

$$h_x = -\frac{2k}{D} \frac{1}{\langle \Theta \rangle} \left. \frac{\partial \Theta}{\partial \tilde{r}} \right|_{\tilde{r}=1}, \quad (2.3)$$

where $\langle \Theta \rangle = (T_1 - \langle T \rangle)/(T_1 - T_0)$. Here, it is always true that $\partial_{\tilde{r}} \Theta \leq 0$. Now, Nu_x can be written as

$$Nu_x = -\frac{2}{\langle \Theta \rangle} \left. \frac{\partial \Theta}{\partial \tilde{r}} \right|_{\tilde{r}=1}, \quad (2.4)$$

where $\langle \Theta \rangle(\tilde{x}/Gz)$. The dimensionless flow-averaged or mixing-cup temperature $\langle \Theta \rangle$ can be calculated according to

$$\langle \Theta \rangle = \frac{\int_0^1 \Theta(\tilde{x}/Gz, \tilde{r}) \tilde{u}(\tilde{r}) \tilde{r} d\tilde{r}}{\int_0^1 \tilde{u}(\tilde{r}) \tilde{r} d\tilde{r}}. \quad (2.5)$$

When $\tilde{x}/Gz > 0.1$, $Nu_x \rightarrow Nu_\infty$.

3 LÉVÊQUE APPROXIMATION

3.1 GOVERNING EQUATION AND VELOCITY PROFILE

To solve the temperature field near the very entrance of the pipe the Lévêque approximation is followed. This involves the following assumptions:

- curvature effects are neglected;
- infinite bulk is assumed;
- the velocity profile is regarded as linear, with the slope given by the slope of the velocity profile at the wall.

Then governing equation can be solved in Cartesian coordinates:

$$u \frac{\partial T}{\partial x} = \alpha \frac{\partial^2 T}{\partial y^2}. \quad (3.1)$$

The wall at $r = R$ is now located at $y = 0$, *i.e.* the directions of the r - and y -axis are reversed.

Under the assumptions listed above, the velocity profile can be written as

$$\tilde{u} = \frac{4}{1 + 4\tilde{b}}(\tilde{y} + \tilde{b}), \quad (3.2)$$

where $\tilde{y} = y/R$.

Three different flow regimes can be distinguished:

1. no-slip, *i.e.* $\tilde{b} = 0$ and hence $\tilde{u} = 4\tilde{y}$;
2. finite slip, *i.e.* $0 < \tilde{b} < \infty$;
3. no-shear, *i.e.* $\tilde{b} = \infty$ and thus $\tilde{u} = 1$.

In the next sections an analytical expression for the local Nusselt number Nu_x is derived for each flow regime. In general,

$$Nu_x \propto \left(\frac{\tilde{x}}{Gz} \right)^{-\beta}. \quad (3.3)$$

The exponent β is constant for the two limiting cases and equals 1/3 for no-slip flow, and 1/2 for no-shear flow. In case of slip flow, $\beta = f(\tilde{x}/Gz, \tilde{b})$.

3.2 NO-SLIP

Making the governing energy equation dimensionless results in the following differential equation for no-slip flow,

$$Gz\tilde{y} \frac{\partial \Theta}{\partial \tilde{x}} = \frac{\partial^2 \Theta}{\partial \tilde{y}^2}, \quad (3.4)$$

with $\Theta(0, \tilde{y}) = 1$, $\Theta(\tilde{x}, 0) = 0$, and $\Theta(\tilde{x}, \infty) = 1$.

By introducing a similarity variable η , where

$$\eta = \left(\frac{Gz\tilde{y}^3}{9\tilde{x}} \right)^{\frac{1}{3}}, \quad (3.5)$$

and subsequent rewriting of the energy balance, we obtain the following ODE:

$$\frac{d^2\Theta}{d\eta^2} + 3\eta^2 \frac{d\Theta}{d\eta} = 0, \quad (3.6)$$

with boundary conditions $\Theta(0) = 0$ and $\Theta(\infty) = 1$. This ordinary differential equation has a known solution, and with the boundary conditions given the final solution for Θ becomes

$$\Theta = \frac{1}{\Gamma(\frac{4}{3})} \int_0^\eta \exp(-\tilde{\eta}^3) d\tilde{\eta}. \quad (3.7)$$

Previously, an expression for the Nusselt number was found using the dimensionless temperature gradient $\partial_{\tilde{y}}\Theta$. Rewriting this in terms of $\partial_{\tilde{y}}\Theta$, the expression for Nu becomes

$$Nu_x = \frac{2}{\langle\Theta\rangle} \left. \frac{\partial\Theta}{\partial\tilde{y}} \right|_{\tilde{y}=0}. \quad (3.8)$$

However, in the L ev eque approximation $\langle\Theta\rangle = 1$. Furthermore, the dimensionless temperature gradient is rewritten in terms of $d_\eta\Theta$. Then

$$Nu_x = \frac{2\eta}{\tilde{y}} \left. \frac{d\Theta}{d\eta} \right|_{\eta=0}. \quad (3.9)$$

Always the temperature gradient $d_\eta\Theta > 0$. Now, using the Leibniz formula for differentiation of integrals, an expression for the temperature gradient can be found:

$$\frac{d\Theta}{d\eta} = \frac{\exp(-\eta^3)}{\Gamma(\frac{4}{3})}. \quad (3.10)$$

Evaluating this at $\eta = 0$ and substituting this in the expression for Nu_x we find

$$Nu_x = \frac{2}{\Gamma(\frac{4}{3})} \frac{\eta}{\tilde{y}}, \quad (3.11)$$

or,

$$Nu_x = \frac{2}{9^{\frac{1}{3}}\Gamma(\frac{4}{3})} \left(\frac{\tilde{x}}{Gz} \right)^{-\frac{1}{3}}. \quad (3.12)$$

Thus, we find that for the no-slip regime the exponent is $\beta = 1/3$.

3.3 NO-SHEAR

For the no-shear case, which implies that we have plug flow, the velocity profile is uniform. Rewriting the governing equation yields the following dimensionless PDE:

$$\frac{Gz}{4} \frac{\partial \Theta}{\partial \tilde{x}} = \frac{\partial^2 \Theta}{\partial \tilde{y}^2}, \quad (3.13)$$

with $\Theta(0, \tilde{y}) = 1$, $\Theta(\tilde{x}, 0) = 0$, and $\Theta(\tilde{x}, \infty) = 1$. Using the similarity variable

$$\eta = \left(\frac{Gz \tilde{y}^2}{16 \tilde{x}} \right)^{\frac{1}{2}}, \quad (3.14)$$

this PDE turns into the following ODE having a known solution,

$$\frac{d^2 \Theta}{d\eta^2} + 2\eta \frac{d\Theta}{d\eta} = 0, \quad (3.15)$$

with $\Theta(0) = 0$ and $\Theta(\infty) = 1$. Solving this ODE gives

$$\Theta = \frac{2}{\sqrt{\pi}} \int_0^\eta \exp(-\tilde{\eta}^2) d\tilde{\eta} = \text{erf}(\eta). \quad (3.16)$$

In order to find an expression for Nu_x , we need an equation for the temperature gradient. This is

$$\frac{d\Theta}{d\eta} = \frac{2}{\sqrt{\pi}} \exp(-\eta^2). \quad (3.17)$$

Ultimately this results in the following expression for Nu_x :

$$Nu_x = \frac{1}{\sqrt{\pi}} \left(\frac{\tilde{x}}{Gz} \right)^{-\frac{1}{2}}. \quad (3.18)$$

Here, $\beta = 1/2$.

3.4 FINITE SLIP

To find an analytical expression for Nu_x for fluid flow with finite slip, we start by rewriting the expression for the velocity profile:

$$u = 4u_{av} \frac{y+b}{R+4b}. \quad (3.19)$$

Plugging this into the governing equation for heat transport, and using the dimensionless variables

$$\Theta = \frac{T_1 - T}{T_1 - T_0}, \quad (3.20)$$

$$Y = \frac{y}{b}, \quad (3.21)$$

$$X = x \frac{\alpha(R+4b)}{4u_{av}b^3}, \quad (3.22)$$

we can non-dimensionalize the governing equation. This yields

$$(1+Y) \frac{\partial \Theta}{\partial X} = \frac{\partial^2 \Theta}{\partial Y^2}, \quad (3.23)$$

with $\Theta(0, Y) = 1$, $\Theta(X, 0) = 0$, and $\Theta(X, \infty) = 1$. To reduce the number of variables, we perform a Laplace transformation of $\partial_X \Theta$ in X ,

$$\mathcal{L}_X \left[\frac{\partial \Theta}{\partial X} \right] = \int_0^\infty \frac{\partial \Theta}{\partial X} e^{-pX} dX = p\bar{\Theta}(p, Y) - \Theta(0, Y) = p\bar{\Theta}(p, Y) - 1, \quad (3.24)$$

where $\bar{\Theta}$ is the Laplace transform of Θ ,

$$\bar{\Theta}(p, Y) = \mathcal{L}_X [\Theta] = \int_0^\infty \Theta(X, Y) e^{-pX} dX. \quad (3.25)$$

Furthermore,

$$\mathcal{L}_X \left[\frac{\partial^2 \Theta}{\partial Y^2} \right] = \frac{\partial^2 \bar{\Theta}}{\partial Y^2}. \quad (3.26)$$

The governing equation now becomes

$$(1+Y)(p\bar{\Theta} - 1) = \frac{\partial^2 \bar{\Theta}}{\partial Y^2}, \quad (3.27)$$

with $\bar{\Theta}(p, 0) = 0$ and $\bar{\Theta}(p, \infty) = 1/p$. To convert this into an ODE with a known solution, we change variable by introducing

$$\hat{\Theta} = \bar{\Theta} - \frac{1}{p}. \quad (3.28)$$

Now we obtain

$$p(1+Y)\hat{\Theta} = \frac{\partial^2 \hat{\Theta}}{\partial Y^2} \quad (3.29)$$

with $\hat{\Theta}(p, 0) = -1/p$ and $\hat{\Theta}(p, \infty) = 0$. Now we change variable a second time by defining

$$\eta = p^{1/3}(1+Y). \quad (3.30)$$

Following Faà di Bruno's formula for second order derivatives saying

$$\frac{\partial^2 \hat{\Theta}}{\partial Y^2} = \frac{\partial^2 \hat{\Theta}}{\partial \eta^2} \left(\frac{\partial \eta}{\partial Y} \right)^2 + \frac{\partial \hat{\Theta}}{\partial \eta} \frac{\partial^2 \eta}{\partial Y^2}, \quad (3.31)$$

we find

$$\frac{d^2 \hat{\Theta}}{d\eta^2} - \eta \hat{\Theta} = 0 \quad (3.32)$$

with $\hat{\Theta}(p^{1/3}) = -1/p$ and $\hat{\Theta}(\infty) = 0$. This ODE is known as the Airy equation, whose general solution of the first kind is the Airy function $\text{Ai}(\eta)$,

$$\text{Ai}(\eta) = \frac{1}{2\pi i} \int_{-i\infty}^{+i\infty} \exp\left(\eta z - \frac{z^3}{3}\right) dz, \quad (3.33)$$

or,

$$\text{Ai}(\eta) = \frac{1}{\pi} \int_0^\infty \cos\left(\eta z + \frac{z^3}{3}\right) dz, \quad (3.34)$$

with $\lim_{\eta \rightarrow \infty} \text{Ai}(\eta) = 0$. Then the solution for $\hat{\Theta}$ becomes

$$\hat{\Theta} = -\frac{\text{Ai}(p^{1/3}(1+Y))}{p\text{Ai}(p^{1/3})}, \quad (3.35)$$

and

$$\bar{\Theta} = \frac{\text{Ai}(p^{1/3}) - \text{Ai}(p^{1/3}(1+Y))}{p\text{Ai}(p^{1/3})}. \quad (3.36)$$

This expression for $\bar{\Theta}$ recovers both $\bar{\Theta}(p, 0) = 0$ and $\bar{\Theta}(p, \infty) = 1/p$. To obtain Θ , we take the inverse Laplace transform in X , giving

$$\Theta(X, Y) = \mathcal{L}_X^{-1}[\bar{\Theta}] = \frac{1}{2\pi i} \int_{c-i\infty}^{c+i\infty} \frac{\text{Ai}(p^{1/3}) - \text{Ai}(p^{1/3}(1+Y))}{p\text{Ai}(p^{1/3})} e^{pX} dp. \quad (3.37)$$

For Nu_x we can derive that

$$Nu_x = \frac{2}{\tilde{b}} \left. \frac{\partial \Theta}{\partial Y} \right|_{Y=0}, \quad (3.38)$$

where, following Leibniz' integral rule,

$$\frac{\partial \Theta}{\partial Y} = -\frac{1}{2\pi i} \int_{c-i\infty}^{c+i\infty} \frac{\text{Ai}'(p^{1/3}(1+Y)) e^{pX}}{p^{2/3} \text{Ai}(p^{1/3})} dp. \quad (3.39)$$

Then, finally, we obtain

$$Nu_x = -\frac{1}{\tilde{b}\pi i} \int_{c-i\infty}^{c+i\infty} \frac{\text{Ai}'(p^{1/3}) e^{pX}}{p^{2/3} \text{Ai}(p^{1/3})} dp, \quad (3.40)$$

or

$$Nu_x = \frac{2}{\tilde{b}} g(X), \quad (3.41)$$

with

$$g(X) = -\frac{1}{2\pi i} \int_{c-i\infty}^{c+i\infty} \frac{\text{Ai}'(p^{1/3}) e^{pX}}{p^{2/3} \text{Ai}(p^{1/3})} dp. \quad (3.42)$$

X turns out to be a function of \tilde{x}/Gz and \tilde{b} , as

$$X = \frac{x}{L} \frac{\alpha L}{4u_{av} R^2} \frac{R^2(R+4b)}{b^3} = \left(\frac{\tilde{x}}{Gz} \right) \frac{1+4\tilde{b}}{\tilde{b}^3}. \quad (3.43)$$

The function $g(X)$ is universal, as it does not depend on the slip length b . The slip length affects the scaling between X and \tilde{x}/Gz , so it determines with part of the function $g(X)$ is relevant. When $\tilde{b} \rightarrow 0$, $X \rightarrow \infty$, while for $\tilde{b} \rightarrow \infty$, $X \rightarrow 0$. As a check, when $\tilde{b} \rightarrow 0$ we expect that

$$g(X) \propto X^{-1/3} \quad (3.44)$$

for large X . Then

$$Nu_x \propto \frac{1}{\tilde{b}} \left(\frac{\tilde{x}}{Gz} \frac{1}{\tilde{b}^3} \right)^{-1/3} = \left(\frac{\tilde{x}}{Gz} \right)^{-1/3}. \quad (3.45)$$

When $\tilde{b} \rightarrow \infty$ we expect that

$$g(X) \propto X^{-1/2} \quad (3.46)$$

for small X . Then

$$Nu_x \propto \frac{1}{\tilde{b}} \left(\frac{\tilde{x}}{Gz} \frac{4\tilde{b}}{\tilde{b}^3} \right)^{-1/2} = \left(\frac{\tilde{x}}{Gz} \right)^{-1/2}. \quad (3.47)$$

To evaluate $g(X)$, let $p = s^3$. Then

$$g(X) = -\frac{1}{2\pi i} \int_{c-i\infty}^{c+i\infty} \frac{\text{Ai}'(s)e^{s^3 X}}{s^2 \text{Ai}(s)} 3s^2 ds = -\frac{3}{2\pi i} \int_{c-i\infty}^{c+i\infty} \frac{\text{Ai}'(s)e^{s^3 X}}{\text{Ai}(s)} ds. \quad (3.48)$$

The integral should start in the sector $-\pi/2 < \arg(s) < -\pi/6$ and finish in the sector $\pi/6 < \arg(s) < \pi/2$. The convergence properties of the integrand are best if we integrate along the rays $\arg(s) = \pm\pi/3$. On the ray $\infty e^{-i\pi/3}$ to the origin we let $s = ke^{-i\pi/3}$ with k going from ∞ to 0. On the ray from the origin to $\infty e^{i\pi/3}$ we let $s = ke^{i\pi/3}$ with k going from 0 to ∞ . Thus

$$g(X) = -\frac{3e^{-i\pi/3}}{2\pi i} \int_{\infty}^0 \frac{\text{Ai}'(ke^{-i\pi/3})e^{-k^3 X}}{\text{Ai}(ke^{-i\pi/3})} dk - \frac{3e^{i\pi/3}}{2\pi i} \int_0^{\infty} \frac{\text{Ai}'(ke^{i\pi/3})e^{-k^3 X}}{\text{Ai}(ke^{i\pi/3})} dk, \quad (3.49)$$

or

$$g(X) = \frac{3e^{-5i\pi/6}}{2\pi} \int_0^{\infty} \frac{\text{Ai}'(ke^{-i\pi/3})e^{-k^3 X}}{\text{Ai}(ke^{-i\pi/3})} dk + \frac{3e^{5i\pi/6}}{2\pi} \int_0^{\infty} \frac{\text{Ai}'(ke^{i\pi/3})e^{-k^3 X}}{\text{Ai}(ke^{i\pi/3})} dk. \quad (3.50)$$

The integrand decays rapidly and can be evaluated numerically. In FIG. 3.1 the function $g(X)$ is plotted as function of X . As demonstrated in the graph, the limiting behaviour of $g(X)$ is as expected: when $X \rightarrow 0$, $g(X) \propto X^{-1/2}$, and when $X \rightarrow \infty$, $g(X) \propto X^{-1/3}$. β , as obtained by computing the gradient of $\log_{10}(Nu_x)$ versus $\log_{10}(\tilde{x}/Gz)$, is plotted in FIG. 3.2. $\beta \approx 5/12$ when $X = 1$. Thus, the dimensionless group X can be considered as a kind of criterion for the behaviour of β : $\beta \rightarrow 1/2$ when $X \ll 1$, whereas $\beta \rightarrow 1/3$ when $X \gg 1$.

The expression

$$G(X) = \frac{0.546 (X^{1/2.420} + 1.058)^{2.420/6}}{X^{1/2}} \quad (3.51)$$

is a good approximation to $g(X)$, which is accurate to about 0.5% for $10^{-4} < X < 10^4$.

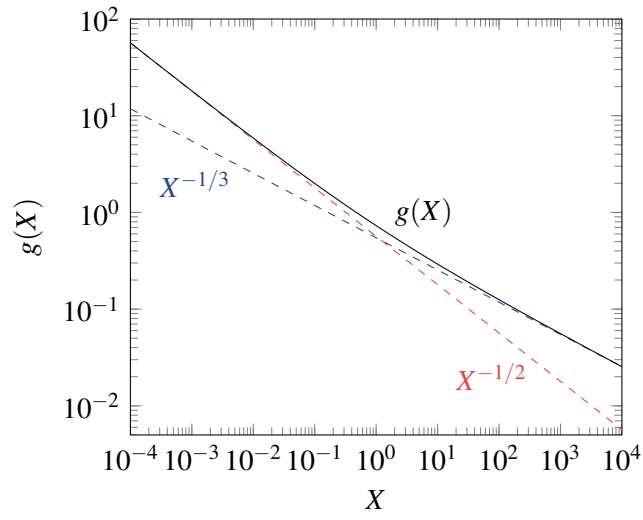


Figure 3.1: $g(X)$ as function of X .

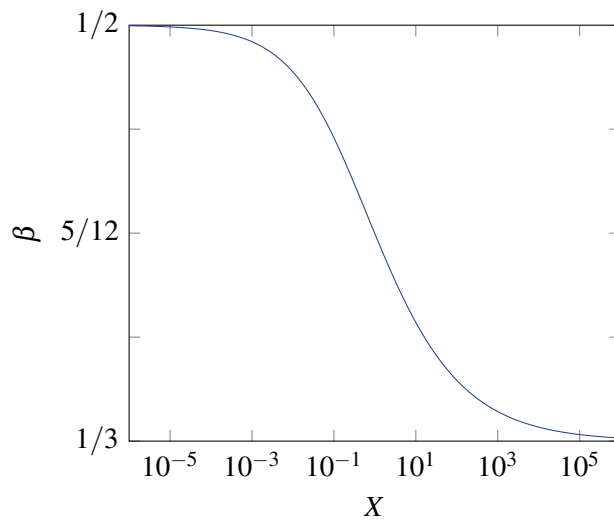


Figure 3.2: β as function of X as obtained from $g(X)$.

4 NUMERICAL PROCEDURE

Numerically, the Graetz-Nusselt problem for finite slip was solved using the pdepe-solver in MATLAB. It is required that the equation to be solved has the following form:

$$c\left(\tilde{r}, \tilde{x}/Gz, \Theta, \frac{\partial \Theta}{\partial \tilde{r}}\right) \frac{\partial \Theta}{\partial (\tilde{x}/Gz)} = \tilde{r}^{-m} \frac{\partial \Theta}{\partial \tilde{r}} \left[\tilde{r}^m f\left(\tilde{r}, \tilde{x}/Gz, \Theta, \frac{\partial \Theta}{\partial \tilde{r}}\right) \right] + s\left(\tilde{r}, \tilde{x}/Gz, \Theta, \frac{\partial \Theta}{\partial \tilde{r}}\right). \quad (4.1)$$

In this study, $c = \tilde{u}/4$, $f = \partial_{\tilde{r}}\Theta$, and $s = 0$. The value of m depends on the coordinate system chosen to solve the equation. For cylindrical coordinates $m = 1$. The initial and boundary conditions were defined according to

$$\Theta(\tilde{r}, (\tilde{x}/Gz)_0) = \Theta_0(\tilde{r}) = 1, \quad (4.2)$$

and

$$p(\tilde{r}, \tilde{x}/Gz, \Theta) + q(\tilde{r}, \tilde{x}/Gz, \Theta) f\left(\tilde{r}, \tilde{x}/Gz, \Theta, \frac{\partial \Theta}{\partial \tilde{r}}\right) = 0. \quad (4.3)$$

For the boundary condition $\Theta(1, \tilde{x}/Gz) = 0$, $p = \Theta(1, \tilde{x}/Gz)$ and $q = 0$. Furthermore, for $\partial_{\tilde{r}}\Theta(0, \tilde{x}/Gz) = 0$, $p = 0$ and $q = 1$. However, boundedness of the solution near $\tilde{r} = 0$ requires that the flux f vanishes at $\tilde{r} = 0$. For $m > 0$, pdepe imposes this boundary condition automatically and it ignores values specified for p and q . The relative and absolute tolerance for the pdepe-solver were set at respectively 10^{-6} and 10^{-12} . For calculating $\partial_{\tilde{r}}\Theta$ at $\tilde{r} = 1$ the MATLAB-function pdeval was utilised.

Because heat transport mainly occurs near the inlet and near the wall, the $\tilde{x} \times \tilde{r} = 82 \times 101$ -mesh was refined near these boundaries. For $\tilde{x} = 0$, the MATLAB logspace-script was used to create an logarithmically equally spaced grid in the x -direction. Refinement in the \tilde{r} -direction was obtained by using a linearly spaced vector (linspace) for \tilde{r}_0 and the following expression:

$$\tilde{r} = 1 - [0.99(1 - \tilde{r}_0^2) + 0.01(1 - \tilde{r}_0)]. \quad (4.4)$$

The values for β_f in the thermally developing regime were obtained by fitting a straight line through $\log_{10}(Nu_x)$ for $-7 \leq \log_{10}(\tilde{x}/Gz) \leq -4$ using the polyfit-algorithm.¹ β_l is the gradient of $\log_{10}(Nu_x)$ or $\log_{10}(g(X))$ versus $\log_{10}(\tilde{x}/Gz)$, computed using the 2nd order accurate gradient-algorithm. Nu_∞ is taken as the average of Nu_x for $\tilde{x}/Gz \geq 0.1$. To compute the flow-averaged or mixing-cup temperature $\langle \Theta \rangle$ the trapz-script was utilised. Gradients of β and Nu_∞ versus \tilde{b} were calculated using the gradient-algorithm.

The thermal boundary thickness $\tilde{\lambda}_T$ was calculated numerically from the temperature profile, which was approximated by linear intrpolation of the temperature to find $\tilde{r}(\Theta = 0.99)$ using two the temperature points closest to $\Theta = 0.99$.

¹Note that changing the range of \tilde{x}/Gz -values used to compute β_f results in a different transition point for β_f . Using larger values shifts the transition point upwards. Nonetheless, the distance between the transition points for β_f and Nu_∞ remains at least one order of magnitude large. When using $-7 \leq \log_{10}(\tilde{x}/Gz) \leq -4$, the transition point for β_f is located at $\tilde{b} = 1.5 \times 10^{-2}$. This point is located at $\tilde{b} = 0.7 \times 10^{-2}$ when using $-7 \leq \log_{10}(\tilde{x}/Gz) \leq -6$. For $-5 \leq \log_{10}(\tilde{x}/Gz) \leq -4$, it is located at $\tilde{b} = 3 \times 10^{-2}$.